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(56) Documents Cited
GB 2128526 A **GB 2123328 A** **GB 2049518 A**
WO 92/18744 A **US 4649777 A** **US 3796418 A**

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INT CL⁶ **E21B**

(54) Abstract Title
Screw connection of pipes

(57) An apparatus for the connection of pipes comprises a rotary (1) and jaws (20, 21, 22), characterised in that all of said jaws (20, 21, 22) are active and are actuatable by pressurised fluid.

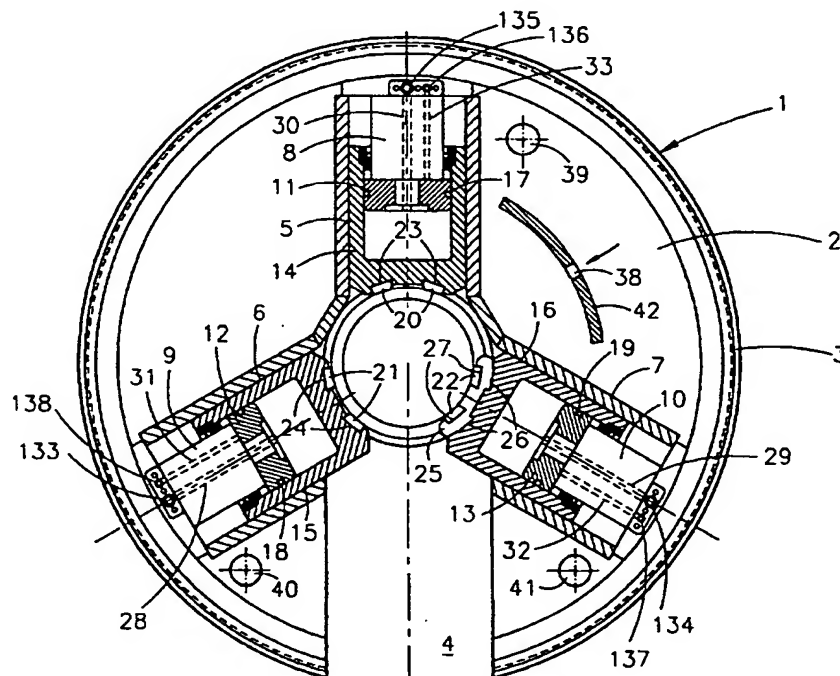


FIG 1

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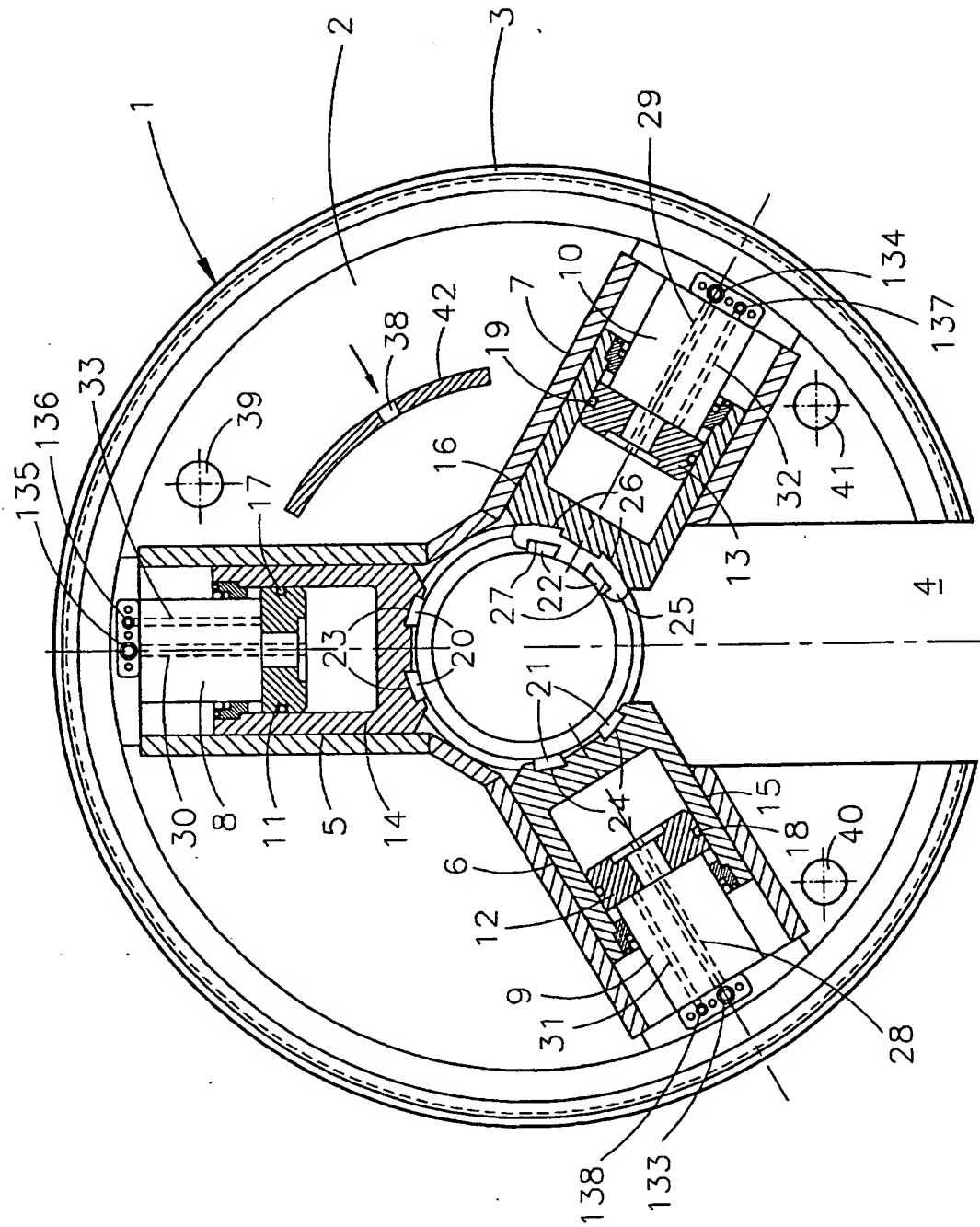


FIG 1

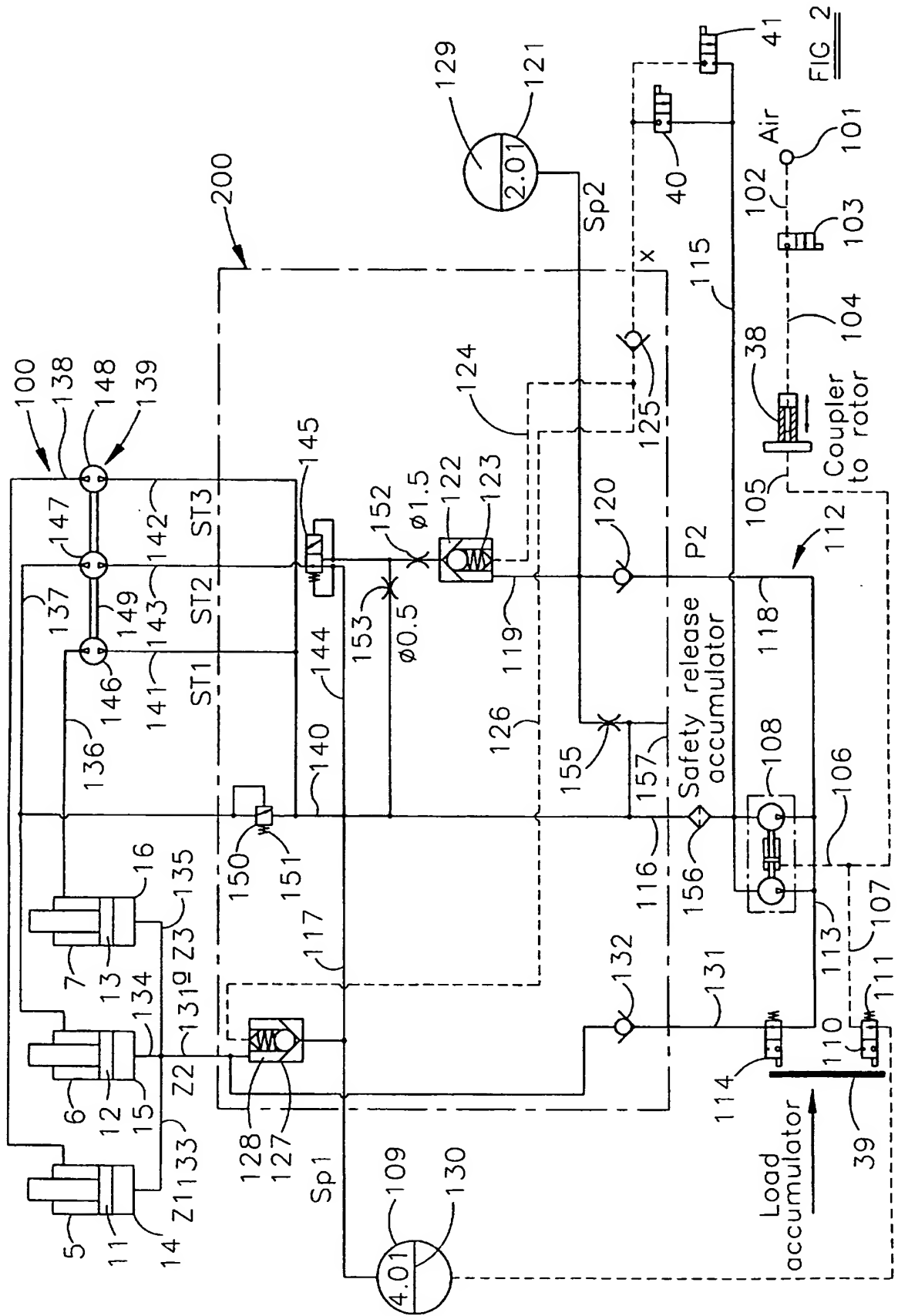


FIG 2

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- 1 -

An Apparatus and a Method for Facilitating the
Connection of Pipes

This invention relates to an apparatus and a method for facilitating the connection of pipes, and more particularly, but not exclusively, to a powered drill
5 pipe tong for facilitating the connection of sections or stands of drill pipe.

Drill pipe tongs are commonly used for facilitating the connection of sections or stands of
10 drill pipe to a pipe string. Typically, the pipe string hangs in a wellbore from a spider in a floor of an oil or gas rig.

A section or stand of drill pipe to be connected to the pipe string is swung in from a drill pipe rack to the
15 well centre above the pipe string. A pipe handling arm may be used to guide the drill pipe to a position above the pipe string. A stabbing guide may then be used to align a threaded pin of the drill pipe with a threaded box of the pipe string. A drill pipe tong is then used to
20 tighten the connection to a torque of typically 68,000Nm (50,000lb.ft).

The drill pipe tong is also used for disconnecting drill pipe. This operation involves breaking the
25 connection which requires a torque typically greater than the tightening torque which may typically be in the order of 110,000Nm (80,000lb.ft).

A drill pipe tong generally comprises jaws mounted in a rotary which is rotatably arranged in a housing. The jaws are moveable relative to the rotary in a generally
30 radial direction towards and away from an upset part of the pipe to be gripped. The upset parts of the pipe are generally located above the pin and below the box of the pipe and have an enlarged outer diameter and/or a reduced inner diameter.

35 In use, the rotary is rotated forcing the jaws along

- 2 -

cam surfaces towards the upset part of the section of pipe. Once the jaws fully engage the upset part, the rotary carries on rotating applying torque to the threads and hence tightens the connection between the section of pipe and the pipe string.

Several problems have been observed with such prior art drill pipe tongs.

In particular, such drill pipe tongs can badly scar the upset part of the pipe, particularly if the jaws start rotating relative to the drill pipe.

Once scarred, the pipe is then lowered into the wellbore. Friction between the wellbore (or casing lining the wellbore) and the scarred upset grinds the upset, reducing the diameter.

Scarring of the upset may also be caused by having to reapply the jaws. This is especially common when connecting pipe with "wedge threads" which requires approximately 80° of turn in order to torque the connection. Many prior art wrenching tongs need to be reapplied to the pipe every 25°.

A reduction in diameter of the upset requires the use of a new drill pipe tong or for the old drill pipe tong to be modified therefor.

An attempt at solving this problem is disclosed in PCT Publication Number WO 92/18744, which discloses a rotary comprising hydraulically operated active jaws and stationary passive jaws. The hydraulically activated jaws are engaged fully with the pipe prior to rotation of the rotary, thereby substantially reducing scarring. A hydraulic circuit is provided on the rotary for actuating the jaws. A plunger is used to activate the hydraulic system by depressing a hydraulic piston of the hydraulic circuit repeatedly. This operation takes time. If several seconds can be saved per connection, the overall cost of the construction of an oil or gas well can be drastically

- 3 -

reduced, as long as reliability is not sacrificed.

Another problem associated with the rotary disclosed in PCT Publication Number WO 92/18744 is that repeated depressing of the plunger for engaging the jaws fully
5 with the pipe may itself cause some scaring.

The present invention provides an apparatus for facilitating the connection of pipes which apparatus comprises a rotary and jaws, characterised in that all of said jaws are active and are actuatable by pressurised
10 fluid.

Other aspects of the invention are set out in claims 2 to 14.

There is also provided a method for facilitating the connection of pipes, the method comprising the steps of
15 actuating the jaws of an apparatus in accordance with the present invention with pressurised fluid so that the jaws engage with a pipe at a central axis of the rotary thereof, rotating said pipe to a predetermined torque, and disengaging said jaws, preferably with said
20 pressurised fluid.

- 4 -

For a better understanding of the invention, reference will now be made, by way of example, to the accompanying drawings, in which:

5 Figure 1 is a top plan view of a rotary of a drill pipe tong in accordance with the invention with parts shown in cross-section; and

Figure 2 is a schematic of a part hydraulic, part pneumatic circuit used in the rotary of Figure 1.

10 Referring to Figure 1 there is shown a rotary which is generally identified by reference numeral 1.

The rotary 1 comprises a rigid housing 2 which is provided with a toothed ring 3 for engagement with toothed drive wheels in a stator of the drill pipe tong (not shown). The housing 2 is also provided with an opening 4 for receiving a drill pipe.

Three piston and cylinders 5, 6 and 7 are arranged about the rotary 1 spaced at 120° to each other and are directed to the centre of the rotary 1. The piston and cylinders 5, 6 and 7 comprise static pistons 8, 9 and 10 each provided with a piston head 11, 12 and 13. Cylinders 14, 15 and 16 are slidable along said piston heads 11, 12 and 13 towards and away from the centre of the rotary 1. Sealing rings 17, 18 and 19 are provided in the piston heads 11, 12 and 13 between the piston heads 11, 12 and 13 and the cylinders 14, 15 and 16.

Cylinders 14, 15 and 16 are provided with jaws 20, 21 and 22 for engaging with the upset of a drill pipe. The jaws 20 and 21 are located in corresponding dovetail slots 23 and 24. The cylinder 16 is shown provided with an extension member 25 between the cylinder 16 and the jaws 22. The extension member 25 is located in dovetail slots 26 and the gripping elements 22 are located in corresponding dovetail slots 27 in the extension member 25. In use, either all of the cylinders 14, 15 and 16

- 5 -

are provided with extension members 25 or none of the cylinders 14, 15 and 16 are provided with extension members 25.

5 Hydraulic lines 28, 29 and 30 and hydraulic lines 31, 32 and 33 are arranged in each piston 8, 9 and 10 for the provision of hydraulic fluid in front of and behind the piston heads 11, 12 and 13.

10 A quick release pneumatic fluid supply connection 38, an accumulator switch 39 and two release switches 40 and 41 are arranged on the housing 2.

15 The quick release pneumatic fluid supply connection 38 is slidably arranged in a slot 42 in the housing 2. The slot 42 is shaped to be concentric with the perimeter of the rotary 1. This allows the rotary 1 to rotate a few degrees with a pneumatic fluid supply line attached.

20 The release switches 40 and 41 are arranged on opposite sides of the rotary so that, when release of the gripping elements 20, 21 and 22 from the drill pipe is required, at least one will be within easy reach of an operator. In particular, in use, part of the stator of the drill pipe tong (not shown) may obscure use of one of the release switches.

25 Referring now to Figure 2 there is shown a circuit which is generally identified by reference numeral 100 arranged in and on the housing 2 of the rotary 1.

30 The circuit 100 is provided with a quick release pneumatic fluid connection 38 slidably arranged in slot 42 of the housing 2 of the rotary 1. The pneumatic fluid is supplied from a source 101 via hose 102, through a valve 103 and through hose 104 to the connection 38. The source supplies pneumatic fluid at approximately 10 bar. A pneumatic line 105 in the housing 2 divides into two branch lines 106 and 107 supplying a pneumatic pump 108 and a bellows 109 respectively. Pneumatic line 107
35 comprises an valve 110 which is biased by spring 111 to

- 6 -

an open position to allow pneumatic fluid to flow to bellows 109.

5 The circuit 100 is charged while the drill pipe tong is situated away from the drill pipe. This step is carried out by moving the valve 103 to an open position to allow pneumatic fluid to flow from source 101 through pneumatic line 105 and by depressing accumulator switch 39. With the accumulator switch 39 depressed, branch line 107 is blocked. Pneumatic fluid actuates pneumatic
10 pump 108, which pumps hydraulic fluid around a sealed circuit 112.

Hydraulic fluid drawn through line 116 and 117 from the bellows 109 is pumped through line 118, through a check valve 120 into an accumulator 121. A line 119 leads
15 from the rear of check valve 120 to a rear side of spring loaded check valve 122. The spring loaded check valve 122 is biased towards a closed position by a spring 123. A control line 124 leads from a rear side of the spring loaded check valve 122, in parallel with spring 123.

20 Since accumulator switch 39 is depressed hydraulic fluid is prevented from being pumped through line 113 by the valve 114 being in a closed position.

Hydraulic fluid is prevented from being pumped through a control line 124 by release valves 40, 41 which
25 are closed and by a check valve 125. Hydraulic fluid is also prevented from being pumped through control line 126 by the check valve 125.

The check valves 120 and 125 inhibit high pressure hydraulic fluid escaping from the accumulator 121.

30 Control line 126 leads from a front side of the check valve 125 to the rear side of a spring loaded check valve 127 in parallel with a spring 128 which bias the spring loaded check valve 127 to a closed position.

Pneumatic fluid 129 in the accumulator 121 is
35 compressed by the pneumatic pump 108 to approximately 280

- 7 -

bar. The pump 108 is prevented from overloading the accumulator by being designed to stall at 280 bar or by use of a pressure relief valve (not shown). The supply of pneumatic fluid is stopped by closing the valve 103.

5 The accumulator switch 39 is now released.

The drill pipe tong can now be offered up to the drill pipe (not shown). The drill pipe is located between the jaws 20, 21 and 22 of the rotary 1 through the opening 4.

10 The jaws 20, 21 and 22 are activated to engage the upset of the drill pipe by opening the valve 103. Pneumatic fluid flows through the valve 103, through line 105 into line 106 and drives the pump 108 and also through line 107 to one side of a membrane 130 in bellows
15 109, squeezing hydraulic fluid to the cylinders 14, 15 and 16 at a high flow rate. Hydraulic fluid pressure acting against spring 128 of the spring loaded check valve 127 opens the spring loaded check valve 127. A small amount of hydraulic fluid is allowed to seep from
20 line 126 past the ball of the spring loaded check valve 122 as it opens.

The pump 108 pumps hydraulic fluid into line 113 through valve 114 into line 131, through a check valve 132 and into the cylinders 14, 15 and 16 via branch lines
25 133, 134 and 135. The pump 108 draws hydraulic fluid from the bellows 109 and from behind the piston heads 11, 12 and 13 through lines 136, 137 and 138, through device 139, through lines 141, 142 into line 140 and through line 143 into line 144 via a flow diverter 145, into line
30 116 into pump 108. The jaws 20, 21 and 22 engage the pipe. The pump 108 will stall or is stopped by removing the pneumatic fluid once the desired engaging force has been reached. This is typically when the pressure in the circuit 100 has built up to 280 bar.

35 It should be noted that, during this procedure, the

- 8 -

accumulator 121 is simultaneously brought up to the same pressure as the engaging pressure if it does not already retain a pressure equal to or higher than the engaging pressure.

5 The flow diverter 145 is biased to allow fluid communication between lines 143 and 144. The device 140 comprises three rotors 146, 147 and 148 arranged on a common shaft 149. When hydraulic fluid flows through the rotors 146, 147 and 148, the rotors allow equal volumes
10 of fluid to pass, thereby ensuring even movement of the jaws 20, 21 and 22 arranged on the cylinders 14, 15 and 16.

The hose 104 may now be disconnected from the connection 38.

15 The rotary 1 may now be rotated to rotate the drill pipe to connect drill pipe.

Once rotation has ceased, the jaws 20, 21 and 22 are disengaged and retracted from the drill pipe. This is carried out by pressing one or both of the release
20 valves 40, 41. This allows hydraulic fluid to flow from the accumulator 121 through control line 124, through spring loaded check valve 122 and through release valves 40 and/or 41 into line 115, line 116 and line 117 to bellows 109. A small amount of hydraulic fluid is allowed
25 to seep past the ball of the spring loaded check valve 122. Hydraulic fluid under pressure also flows from control line 126, allowing pressurised hydraulic fluid to flow from in front of the piston heads 11, 12 and 13 to bellows 109. High pressure hydraulic fluid shifts flow
30 diverter 145 allowing high pressure hydraulic fluid to flow into line 143. The flow through line 143 rotates the rotor 147, which rotatably drives rotors 146 and 148 via shaft 149 and sucks hydraulic fluid out of bellows 109 into the cylinders behind the piston heads 11, 12 and 13
35 and retracts the jaws 20, 21 and 22 in unison. A valve

150 is arranged in parallel with line 143 and bypasses the device 139. The valve 150 is biased by a spring 151 to a closed position, however upon the pressure increasing on the rear side of the piston head 12, the
5 valve 150 opens equalling the flow rate between the driving rotor 147 and the driven rotors 146 and 148.

The hydraulic fluid in front of the piston heads 11, 12 and 13 is expelled through branch lines 133, 134 and 135 into line 131a and passes through spring loaded check
10 valve 128 into line 117 and into bellows 109. The residual hydraulic fluid due to the difference in volumes of the cylinders 14, 15 and 16 when engaged and retracted, flows is stored in the bellows 109. Restrictors 152 and 153 inhibit sudden changes in
15 pressure upon depression of the release valve 40, 41 and the opening of spring loaded check valve 122.

A safety release valve 155 is provided such that if pressure in the accumulator 121 needs to be released the safety valve can be operated to vent the hydraulic fluid
20 to atmosphere or into a safety release accumulator 156. The safety release valve may be operated by a control or be a removable cap in a block 200.

The valves 120, 122, 125, 127, 132, 145, 155 and the respective lines and control lines are arranged in a
25 single block 200. The block 200 may be formed from any suitable material such as aluminium, aluminium alloy or steel.

It should be noted that the entire circuit 100 is arranged in or/and on the rotary 1. The pneumatic fluid
30 source 101 is of the type provided on most drilling rigs and is typically at a pressure of 10 bar.

Various modifications are envisaged to the above apparatus. In particular, it is envisaged that a further accumulator could be provided for providing a charge for
35 moving the jaws into engagement with a pipe. This has

- 10 -

the advantage that the pneumatic fluid line may be removed from the drill pipe tong before the drill pipe tong is moved about the pipe thus saving vital seconds disconnecting the hose from the rotary.

- 5 It is also envisaged that the apparatus could be used with thin walled pipe, as it is relatively simple to alter the force applied to the pipe by the jaws.

It is also envisaged that the accumulator could take the form of a spring or a battery

CLAIMS

1. An apparatus for facilitating the connection of pipes which apparatus comprises a rotary (1) and jaws
5 (20, 21, 22), characterised in that all of said jaws (20, 21, 22) are active and are actuatable by pressurised fluid.
2. An apparatus as claimed in Claim 1, wherein each of said jaws (20,21,22) is mounted on the cylinder of a respective piston and cylinder (5,6,7).
- 10 3. An apparatus as claimed in Claim 2, wherein the cylinders (14,15,16) of said at least two piston and cylinders (5,6,7) move towards said pipe to engage therewith.
4. An apparatus as claimed in Claim 3, wherein each of
15 said cylinders (14,15,16) comprises a curved end to substantially conforms to said pipe.
5. An apparatus as claimed in Claim 1, 2, 3 or 4 wherein said jaws (20, 21, 22) further comprising pipe engaging inserts.
- 20 6. An apparatus as claimed in Claim 5, wherein said pipe engaging inserts (20,21,22) are angled to be normal to said pipe.
7. An apparatus as claimed in Claim 5 or 6, wherein said pipe engaging inserts (20,21,22) are those sold by
25 the applicants under the Trade Mark MICRO-GRIP.
8. An apparatus as claimed in any preceding claim wherein said rotary (1) comprises an open, radially extending, passageway (4) to facilitate the passage of a pipe into the centre of said rotary (1).
- 30 9. An apparatus as claimed in any of Claims 2 to 8, further comprising a device (139) to help ensure that said jaws (20,21,22) move in unison.
10. An apparatus as claimed in Claim 9, wherein said device (139) comprises a rotor (146,147,148) for each of
35 said jaws (20,21,22) and a common shaft (149) to which

- 12 -

said rotors are fixed.

11. An apparatus as claimed in any of Claim 2 to 10, further comprising extension members (25) for extending the reach of said piston and cylinders (5,6,7).

5 12. An apparatus as claimed in any of Claims 2 to 11 comprising three piston and cylinders (5,6,7).

13. A power tong incorporating an apparatus as claimed in any preceding claim.

10 14. A method for facilitating the connection of pipes, said method comprising the steps of actuating the of an apparatus as claimed in any of Claims 1 to 11, or a power tong as claimed in Claim 12 with pressurised fluid, so that the jaws engage with a pipe at a central axis of the rotary, rotating said pipe to a predetermined torque, and
15 disengaging said jaws.

15. A method as claimed in Claim 14, wherein said jaws are disengaged with said pressurised fluid.



Application No: GB 9901778.2
Claims searched: 1-15

Examiner: Dave Butters
Date of search: 20 April 1999

Patents Act 1977
Search Report under Section 17

Databases searched:

UK Patent Office collections, including GB, EP, WO & US patent specifications, in:

UK CI (Ed.Q): B3N

Int CI (Ed.6): E21B

Other:

Documents considered to be relevant:

Category	Identity of document and relevant passage	Relevant to claims
X,Y	GB 2128526 A (CHRISTENSEN)	X-1 Y-2,3,8, 13,14,15
X,Y	GB 2123328 A (MACHINE TOOL)	X-1 Y-2,3,8,9, 13,14,15
X,Y	GB 2049518 A (ECKEL)	X-1 Y-2,3,4,8
Y	WO 92/18744 A (WEATHERFORD)	2,3,8,9,13 14,15
X	US 4649777 A (BUCK)	1-8,12-15
X	US 3796418 A (JACKSON)	1-7,11-14

X Document indicating lack of novelty or inventive step
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